Entanglement-enhanced optomechanical sensor array for dark matter searches

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The nature of dark matter is one of the most important open questions in modern physics. The search for dark matter is challenging since, besides gravitational interaction, it feebly interacts with ordinary matter. Here, we propose entanglement-enhanced optomechanical sensing systems to assist the search for DM with mechanical sensing devices. To assess the performance of our setup, we adopt the integrated sensitivity, which is particularly suitable for broadband sensing as it precisely quantifies the bandwidth-sensitivity tradeoff of the system. We then show that, by coherently operating the optomechanical sensor array and utilizing continuous-variable multipartite entanglement between the optical fields, the array of sensors has a scaling advantage over independent sensors (i.e., $\sqrt{M} \rightarrow M$, where M is the number of sensors) as well as a performance boost due to entanglement. Such an advantage is robust to imhomogeneities of the mechanical sensors and is achievable with off-the-shelf experimental components.

Introduction.— Identifying the nature of Dark Matter (DM) is one of the most pressing quests for fundamental physics research. The evidence for the existence and the particle nature of DM are ubiquitous—such as the cosmic microwave background survey [1, 2], gravitational lensing [3] and rotation curves of spiral galaxies [4–7] with a consistent DM mass ranging over eighty orders of magnitude. Many searches focus on the particle-like regime, such as weakly interacting massive particles via direct detection searches. On the other hand, exploring the sub-eV regime for wave-like DM poses interesting experimental challenges and new opportunities for particle physics discovery.

Depending on the DM model, various types of DM sensors have been designed. In axion DM model, DM can induce photons in a background magnetic field; therefore microwave cavities immersed in a powerful magnetic field are leveraged for a DM search [8–18]. In other models, DM induces forces on normal matter; therefore mechanical sensors good at sensing weak forces can be used for a DM search [18–27]. These new potential couplings could reveal how the relic density of DM comes about and possibly explain other puzzles of fundamental physics, such as the strong CP puzzle, the baryon-antibaryon asymmetry of our universe, and many others. As the precision of sensors approaches the quantum regime, shot noise from vacuum fluctuations becomes relevant. To overcome vacuum noise, in the microwave cavity case, a squeezing enhanced DM search has been proposed [14, 15] and demonstrated [17]. Recently, arrays of microwave cavities [28–31] and optomechanical sensors [22, 32, 33] have been proposed to further enhance DM searches. The benefits of coherent post-processing with an optomechanical array operating at the standard quantum limit (SQL) have been discussed [22, 32, 33], but explicit details—as well as potential advantages from entanglement-enhanced readout—remain largely unexplored. With backaction into play, it is unclear whether an optomechanical array can enjoy the entanglement advantages previously shown in the microwave case [31].

In this work, we propose entanglement-enhanced readout of mechanical sensors for a DM search (see Fig. 1), exploiting recently developed techniques in distributed quantum sensing (DQS) [34–37]. Building upon the quantum theory of optomechanics [38–40], we show that, by coherently operating an array of mechanical sensors and utilizing continuous-variable multi-partite entanglement between the optical fields, entanglement enhancement and advantageous scaling with the number of sensors are simultaneously achievable.

Dark matter model.— The basic model of the driving force exerted on the mechanical oscillator (in the surface normal direction) by DM, with the mass density $\rho_{\rm DM}$, can be described by $F_{\rm dr}(t) \simeq F_{\rm DM} \cos(\Omega_{\rm DM}t + \varphi)$ within a coherence time, with the amplitude of the drive force

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Figure 1. Illustration of our DQS proposal. A stochastic field (e.g., DM field; background cloud) faintly drives an array of mechanical oscillators, in a correlated manner, leading to feeble vibrations of the oscillators' positions. To interrogate the oscillators, squeezed laser light is distributed to the array via passive elements (beam-splitters, described by weights $\{w_{j0}\}$), which generates an entangled state. The impinging radiation reflects off the oscillators and is detected via homodyne detection. The measurement results are jointly combined via the weights $\{W_{j0}\}$, from which the signal is inferred.

 $F_{\rm DM} = g \sqrt{\rho_{\rm DM}} \mathcal{M}$, where g quantifies the coupling of DM to the mechanics for a particular DM model and \mathcal{M} characterizes material and geometrical properties of the sensor [21, 22, 41]. For the DM frequencies under consideration $\Omega_{\rm DM} \sim \text{kHz}$ (representing the dark matter particle's mass), the de Broglie wavelength of DM is around 10^5 km; this large scale is pertinent for array-based setups that wish to take advantage of the signal correlations between the sensors within the array. Overall, the phase φ is random and therefore $\hat{F}_{\rm dr}(t)$ is described by a stationary random process with a coherence time $1/\Delta_a$, and can be characterized by its power spectral density (PSD) $S_{F_{\rm dr}} \sim F_{\rm DM}^2/\Delta_a$. For any time-dependent operators \hat{O}, \hat{O}' with stationary statistics, the PSD is defined as

$$S_{\hat{O}\hat{O}'}(\omega) = \frac{1}{2\pi} \int_{-\infty}^{+\infty} \mathrm{d}\omega' \left\langle \hat{O}^{\dagger}(-\omega)\hat{O}'(\omega') \right\rangle, \quad (1)$$

where $\hat{O}^{\dagger}(\omega)$ is the Fourier transform of the time-domain operator. We also define a symmetrized PSD, $\bar{S}_{\hat{O}\hat{O}'}(\omega) \equiv [S_{\hat{O}\hat{O}'}(\omega) + S_{\hat{O}\hat{O}'}(-\omega)]/2$. When $\hat{O} = \hat{O}'$, we simplify the notation as $S_{\hat{O}}(\omega)$.

Simplified model. — For clarity, we start our discussion with a simplified model for optomechanical sensors and then proceed to the full cavity model [38–40]. As shown in Fig. 1, each optomechanical sensor system is composed of a highly reflective mechanical oscillator which couples to a free electromagnetic field. DM hypothetically couples to the mechanics and drives the oscillator's motion.

To detect the motion of the oscillator, one stimu-

lates an input light field $\hat{E}^{in}(t)$, which impinges on the mechanics, and measures the output field $\hat{E}^{out}(t)$ postinteraction. The mechanics completely reflects the input field and induces a feeble phase shift, $\zeta \hat{q}(t) \ll 1$, such that $\hat{E}^{in}(t) \rightarrow e^{i\zeta \hat{q}(t)} \hat{E}^{in}(t)$, where $\zeta = 2\Omega_L/c$ and \hat{q} is the position operator of the mechanics in meters.

We decompose the input field operator at the source in the frequency domain $\hat{E}^{in}(t) = e^{-i\Omega_L t} \Big[E_0 + \int \frac{d\omega}{2\pi} \hat{a}^{in}(\omega) e^{-i\omega t} \Big]$, where E_0 is the amplitude of the laser, and we factored out the fast oscillations due to the carrier frequency, Ω_L . Here, $\hat{a}^{in}(\omega)$ is the modal annihilation operator with zero-mean, satisfying the commutation relation $[\hat{a}^{in}(\omega), \hat{a}^{in\dagger}(\omega')] = 2\pi\delta(\omega - \omega')$. For later use, we define $\hat{X}^{in}(\omega) \equiv (\hat{a}^{in}(\omega) + \hat{a}^{in\dagger}(-\omega))/\sqrt{2}$ and $\hat{Y}^{in}(\omega) \equiv (\hat{a}^{in}(\omega) - \hat{a}^{in\dagger}(-\omega))/\sqrt{2}i$, which are power spectral amplitudes of the amplitude and phase quadratures, $\hat{X}^{in}(t)$ and $\hat{Y}^{in}(t)$, respectively, of the optical field.

The output (including, e.g., detector efficiency, $0 \leq \eta^2 \leq 1$) can be expressed to leading order as $\hat{E}^{\text{out}}(t) \approx \eta \Big[E_0(1 + \mathrm{i}\zeta \hat{q}(t) + \int \frac{\mathrm{d}\omega}{2\pi} \hat{a}^{\text{in}}(\omega) e^{-\mathrm{i}\omega t} \Big] + \cdots$, where loss-induced vacuum terms and the overall factor $e^{-\mathrm{i}\Omega_L t}$ are omitted for simplicity. The phase quadrature of the output field is measured via homodyne detection with a local oscillator field of the same carrier frequency. We calculate its spectral amplitude directly as,

$$\hat{Y}^{\text{out}}(\omega) = \sqrt{2}\eta \Big[\hat{Y}^{\text{in}}(\omega) + \sqrt{2}E_0\zeta \hat{q}(\omega) \Big] + \cdots \qquad (2)$$

where we omit a constant pre-factor from the amplitude of the local oscillator. Overall, the motion of the oscillator leads to a detectable displacement on the quadrature.

Cavity optomechanics.— In order to analyze the setup in a more complete fashion, we present a full optomechanical cavity model [38–40] and comment how the simplified model above corresponds to the theory of cavity optomechanics in the bad cavity limit.

We utilize linear input-output theory to describe radiation coupling to an optical cavity with one vibrating mirror (the mechanics). In this theory, the intracavity field and the mechanical motion of the mirror are dissipatively coupled to ingoing bath-modes— $(\hat{X}^{\text{in}}, \hat{Y}^{\text{in}})$ and $(\hat{Q}^{in}, \hat{P}^{in})$ —at dissipation rates κ and γ , respectively. The equations of motion for the open quantum system lead to a set of coupled first-order differential equations in the time domain for the intra-cavity modes (X, Y, \hat{Q} , and \hat{P} in terms of the coupling rates and the ingoing bath modes, which can be analytically solved in the frequency domain. Here, $\hat{Q} = \hat{q}/\sqrt{2}q_{zp}$ is the normalized position operator, where $q_{\rm zp} \equiv \sqrt{\hbar/2m\Omega}$ is the zero-point motion, and \hat{P} is the conjugate momentum. The outgoing fluxes—denoted by the operators $(\hat{X}^{\text{out}}, \hat{Y}^{\text{out}})$ and $(\hat{Q}^{\text{out}}, \hat{P}^{\text{out}})$ —are then determined via the input-output relations $\hat{X}^{\text{out}} = \hat{X}^{\text{in}} - \sqrt{\gamma} \hat{X}$ etc.; see Appendix D and Refs. [38, 39] for further details.

An exact output relation for the spectral amplitude of

the phase quadrature can be found [38, 39],

$$\hat{Y}^{\text{out}}(\omega) = -e^{i\varphi_{\omega}}\hat{Y}^{\text{in}}(\omega) + 2\sqrt{2\gamma C_{\omega}}\hat{Q}(\omega), \qquad (3)$$

where the phase φ_{ω} and the optomechanical cooperativity C_{ω} are defined via

$$e^{i\varphi_{\omega}} \equiv \left(\frac{\kappa/2 + i\omega}{\kappa/2 - i\omega}\right), C_{\omega} \equiv \frac{2G^2/\gamma\kappa}{(1 - 2i\omega/\kappa)^2} = |C_{\omega}|e^{i\varphi_{\omega}}.$$
(4)

Here $G \equiv EG_0$ is the cavity-enhanced optomechanical coupling-rate, G_0 is the vacuum optomechanical coupling rate, and E is the intra-cavity field (taken to be real). The intra-cavity field, E, is related to the amplitude of the input field, E_0 , via $E^2 = (4\kappa_r/\kappa^2)E_0^2$, where κ is the total dissipation-rate of the cavity and κ_r is the dissipation-rate to the readout port. Here, the spectral amplitude of the oscillator's position, $\hat{Q}(\omega)$, is given as,

$$\hat{Q}(\omega) = 2\sqrt{\gamma}\chi_{\omega} \left(\hat{P}^{\rm in}(\omega) - \sqrt{2C_{\omega}}\hat{X}^{\rm in}(\omega)\right) + Q_{\rm dr}(\omega), \quad (5)$$

where $Q_{\rm dr}(\omega)$ is the diplacement induced by the driving force, $F_{\rm dr}$, and is related via $F_{\rm dr}(\omega) \equiv (\sqrt{\hbar m \Omega}/\chi_{\omega}) Q_{\rm dr}(\omega)$ and $\chi_{\omega} = \Omega/(\Omega^2 - \omega^2 - i2\gamma\omega)$ is the mechanical susceptibility. The term proportional to the amplitude quadrature, $\hat{X}^{\rm in}$, represents the fluctuation of the oscillator's position due to radiation pressure.

Indeed, with the complete cavity optomechanics model, we recover the mechanical motion induced quadrature displacement identified in Eq. (2) in the simplified model. The relation can be made quantitative by treating the output mirror in the cavity model as a transparent window, see Appendix D for further comparisons.

We estimate the force impressed on the mechanics from homodyne measurements on the phase-quadrature via

$$\hat{F}(\omega) \equiv \frac{\mathrm{e}^{-\mathrm{i}\varphi_{\omega}/2}}{\chi_{\omega}} \sqrt{\frac{\hbar m\Omega}{8\gamma |C_{\omega}|}} \hat{Y}^{\mathrm{out}}(\omega), \qquad (6)$$

which has units N/\sqrt{Hz} . A general expression for the noise spectrum can also be derived,

$$\bar{S}_{F_{\text{noise}}}(\omega) = \frac{\hbar m \Omega}{8\gamma |C_{\omega}| |\chi_{\omega}|^2} \bar{S}_{\hat{Y}^{\text{in}}}(\omega) + 8\hbar m \gamma \Omega |C_{\omega}| \bar{S}_{\hat{X}^{\text{in}}}(\omega) + \frac{2\hbar m \Omega}{|\chi_{\omega}|} \operatorname{Re}\left(\frac{\chi_{\omega}}{|\chi_{\omega}|} \tilde{S}_{\hat{X}^{\text{in}}\hat{Y}^{\text{in}}}(\omega)\right) + 4\hbar m \gamma \Omega \bar{S}_{\hat{P}^{\text{in}}}(\omega), \quad (7)$$

where we have defined,

$$\tilde{S}_{\hat{X}^{\text{in}}\hat{Y}^{\text{in}}}(\omega) \equiv \frac{S_{\hat{X}^{\text{in}}\hat{Y}^{\text{in}}}(\omega) + S^*_{\hat{X}^{\text{in}}\hat{Y}^{\text{in}}}(-\omega)}{2}.$$
 (8)

The first term in Eq. (7) is the shot noise, the second term is the back-action noise due to radiation pressure, the third term encodes the quadrature correlations, and the fourth term consists of mechanical fluctuations—e.g., $\bar{S}_{P^{\text{in}}} \approx K_B T/\hbar\Omega$ for thermally dominated fluctuations. The SQL can be obtained by assuming initial vacuum fluctuations $(\bar{S}_{\hat{Y}^{\text{in}}} = \bar{S}_{\hat{X}^{\text{in}}} = 1/2$ and $\tilde{S}_{\hat{X}^{\text{in}}\hat{Y}^{\text{in}}} = 0$) and choosing $|C_{\omega}| = 1/8\gamma |\chi_{\omega}|$, then (ignoring mechanical noise) the noise at the SQL is $\bar{S}_{F_{\text{noise}}}^{\text{SQL}} \equiv \hbar m \Omega / |\chi_{\omega}|$. We can also incorporate detection loss $1 - \eta^2$ in the cavity model via the simple substitution $\bar{S}_{F_{\text{noise}}} \to \bar{S}_{F_{\text{noise}}} + \frac{1-\eta^2}{\eta^2} (\hbar m \Omega / 16\gamma |C_{\omega}| |\chi_{\omega}|^2)$; see Appendix D for more discussion on loss.

Quantifying performance. — Since the mass of the DM signal is a priori unknown, one must integrate over many frequencies to rule out a range of potential masses for DM [8–13, 16, 17]. Hence, detection bandwidth of the setup is paramount, however sensitivity is equally important, as such is needed to quickly build statistical confidence in our measurements. A general figure of merit for broadband sensing of an incoherent force, which takes both sensitivity and bandwidth into account, is the integrated sensitivity,

$$\mathcal{I}_{\Omega} \equiv \int_{0}^{\infty} \left(\frac{\bar{S}_{F_{\rm dr}}(\omega)}{\bar{S}_{F_{\rm noise}}(\omega)} \right)^{2} \frac{\mathrm{d}\omega}{\pi}.$$
 (9)

For a thorough discussion about the integrated sensitivity being a good figure of merit in DM searches, see Refs. [42, 43] and Appendix B 2 for further discussion. Later, we evaluate the integrated sensitivity for an array of M sensors and denote the quantity as $\mathcal{I}_{\Omega}^{(M)}$. Assuming $\bar{S}_{F_{\rm dr}}$ is approximately flat over the integra-

Assuming $\bar{S}_{F_{\rm dr}}$ is approximately flat over the integration range (e.g., due to no prior information about DM mass [44]), we find the integrated sensitivity for the SQL, $\mathcal{I}_{\Omega}^{\rm SQL}/\bar{S}_{F_{\rm dr}}^2 = 4\gamma/(\hbar m \Omega \gamma)^2$, which is the ratio of the mechanical linewidth, γ , and the on-resonance sensitivity at the SQL, $\bar{S}_{F_{\rm noise}}^{\rm SQL}(\Omega) = \hbar m \Omega \gamma/2$. Without quantum resources, this sets the ultimate classical limit in broadband detection for a given set of mechanical parameters m, γ , and Ω , which in turn imposes limits on a DM search with a mechanical system.

Squeezing enhanced performance.— Squeezing the input radiation is known to increase the effective bandwidth in optomechanical sensing while leaving the peak sensitivity (set by the SQL) the same [46–52], thus resulting in squeezing-enhanced broadband sensing. We briefly analyze such for a single mechanical sensor, utilizing the integrated sensitivity as a performance metric; see Appendix D for more discussion on squeezing. In Fig. 2, we plot the integrated sensitivity, \mathcal{I}_{Ω} , versus the mechanical resonance frequency, Ω , for various detector configurations (classical and squeezed setups), assuming an approximately flat spectrum for the signal, \bar{S}_{Fdr} .

We incorporate different optimization schemes in our analysis. The values E^{\star}_{ω} and θ^{\star}_{ω} (where E is the field amplitude and θ is the squeezing angle) in Fig. 2 are chosen via $(E^{\star}_{\omega}, \theta^{\star}_{\omega}) = \operatorname{argmax}[\bar{S}_{\hat{F}_{dr}}/\bar{S}_{F_{noise}}]$, whereas \tilde{E}^{\star} and $\tilde{\theta}^{\star}$ are chosen via $(\tilde{E}^{\star}, \tilde{\theta}^{\star}) = \operatorname{argmax}[\mathcal{I}_{\Omega}]$. The former optimum (which is frequency dependent) may not always be practically feasible. If the squeezing angle,



Figure 2. Single-sensor, integrated sensitivity for various detector configurations (with and without squeezing) versus the mechanical resonance frequency, Ω . The Classical Limit (CL) is set by the SQL (which optimizes the power for each frequency ω) plus mechanical thermal fluctuations. The Practical CL operates with a fixed power, independent of frequency ω . Squeezing level is 4.2 dB (equivalent $N_s = .25$ squeezed photons) for each squeezed curve; θ is the squeezing angle. Optomechanical parameters are T = 10 K, $\mathcal{Q} \equiv \Omega/2\gamma = 10^9$, $\kappa = 2\pi \times 16$ MHz, m = 10 ng, and $G_0 = 2\pi \times 121$ Hz (cf. Refs. [20, 45]). Here, $(E_{\omega}^{\star}, \theta_{\omega}^{\star}) = \operatorname{argmax}[\bar{S}_{\bar{F}_{dr}}/\bar{S}_{F_{noise}}]$ and $(\tilde{E}^{\star}, \tilde{\theta}^{\star}) = \operatorname{argmax}[\mathcal{I}_{\Omega}]$.

 θ , of the input radiation is tunable, such that $\theta = \theta_{\omega}^{\star}$ may be chosen, then squeezing can surpass the Classical Limit (CL, which is set by the SQL plus mechanical thermal noise) on broadband detection sensitivity; see Refs. [49–52], where tunability of the squeezing angle in optomechanical systems is addressed.

Optomechanical sensor array.— We now present our main contribution, where we extend the results on squeezing-enhanced broadband sensitivity to entanglement-enhanced broadband sensitivity with an array of M mechanical sensors. In our setup, the optomechanics are not directly coupled across the array; rather, we allow for mixing of the input and output optical fields via linear optical elements (Fig. 1). We further suppose that the stochastic drive force (e.g., the DM field) impresses a correlated displacement on the sensors. We show that, by utilizing entangled optical fields to measure the mechanics, the squeezing-enhancement demonstrated for a single sensor naturally extends to a sensorarray with the same amount of squeezed photons.

Consider M input modes, $\{\hat{a}_n^{\text{in}}\}_{n=0}^{M-1}$, and a strong laser field at frequency Ω_L on the \hat{a}_0^{in} mode, with non-trivial quantum fluctuations (e.g., squeezing) on the sidebands. This mode is mixed with the idling input-modes (consisting of uncorrelated vacuum fluctuations) of the remaining M-1 inputs via linear optical elements described by the dividing-weights $\{w_{n0}\}$, with $w_{n0} \in \mathbb{C}$. After interaction with the mechanics, we measure the outgoing phase quadrature at each sensor, \hat{Y}_n^{out} , via homodyne detection (see Appendix E for more details). In post-processing, we convert the measurement result at the *n*th sensor to a force measurement via the relation (6) and write the resulting value as \hat{F}_n . We then statistically combine the signals from each sensor with combining-weights $\{W_{0n}\}$, with $W_{0n} \in \mathbb{C}$, and construct a weighted average force estimator,

$$\hat{\overline{F}}(\omega) \equiv \sum_{n=0}^{M-1} W_{0n} \hat{F}_n(\omega).$$
(10)

In this manner, we capitalize on the correlations of the stochastic drive field (e.g., the spatial-uniformity of the DM field) across the array to achieve favorable scaling with the size of the array.

Suppose that the drive force obeys the following statistics, $\langle \hat{F}_{\mathrm{dr},n}\hat{F}_{\mathrm{dr},n'}\rangle = \mathcal{M}_n\mathcal{M}_{n'}f^2$, where f is common to each sensor and \mathcal{M}_n is a sensor dependent pre-factor (e.g., $f \sim \sqrt{\rho_{\mathrm{DM}}/\Delta_a g}$ for DM). The signal PSD of the force is then,

$$\bar{S}_{\hat{F}_{\rm dr}}^{(M)}(\omega) = \left|\sum_{n=0}^{M-1} W_{0n} \mathcal{M}_n\right|^2 f^2.$$
(11)

Consider the ideal scenario where all the sensors are identical. In this case, the dividing- and combining-weights are chosen to satisfy $\sum_k W_{0n}^* w_{k0} = \delta_{nk}$ and $|w_{k0}| = |W_{k0}| = 1/\sqrt{M}$. In words, since the performance of each sensor is identical and the response of each sensor to the drive force is identical, the best strategy is to distribute the input radiation uniformly to each sensor and then uniformly combine the signals.

The signal PSD in the ideal setting is M times the single-sensor signal PSD $(\bar{S}_{\hat{F}_{\rm dr}}^{(M)} = M\bar{S}_{\hat{F}_{\rm dr}})$ which follows directly from Eq. (11). This is due to the classical correlations of the drive field across the array. Moreover, if we scale the laser power with the number of sensors, $E_0^2 \to M E_0^2$, such that the power per sensor is held fixed as we increase the number of sensors, then the multisensor force noise [Eq. (E24) of Appendix E] reduces to the single-sensor force noise of Eq. (7). Therefore, it follows that we can use the same amount of squeezing as in the single sensor setting to achieve an equivalent broadband reduction of the noise in the multi-sensor setup, which comes along with the aforementioned M-factor boost to the signal.

Our results are captured in Fig. 3, where we plot the integrated sensitivity for an array of identical sensors versus the number of sensors. The power per sensor is chosen to optimize the SNR at each frequency (i.e., $E = E_{\omega}^{\star}$ as in Fig. 2, such that $E_{\text{total}}^2 = M E_{\omega}^{\star 2}$), prior to integrating over all frequencies. For the DQS setup, a squeezed vacuum, with $N_s = .25$ squeezed photons and squeezing angle $\theta = \theta_{\omega}^{\star}$, is distributed uniformly across the array. We observe scaling enhancements for both Distributed Classical Sensing (DCS) and DQS setups. The



Figure 3. Scaling of the integrated sensitivity versus the number of mechanical sensors (with resonance frequencies $\Omega = 10 \mathrm{kHz}$) in the array. Independent sensors operate at the SQL (plus mechanical fluctuations) with no joint postprocessing. CL-DCS scheme operates each sensor at the SQL (plus mechanical fluctuations) and leverages joint postprocessing. DQS scheme leverages joint post-processing and entanglement by distributing a squeezed vacuum (of $N_s = .25$ total squeezed photons) uniformly to the sensors. Inset shows robustness of performance for two sensors (M = 2) versus the amount of inhomogeneity (parameterized by $0 \le p \le 1$) in the array. Optomechanical parameters take on the same values as in Fig. 2. The input power is optimized at the single-sensor level, such that the total input power to the array is $E_{\text{total}}^2 = M E_{\omega}^{*2}$.

CL-DCS setup—which operates each sensor at the SQL (plus mechanical fluctuations) and leverages joint postprocessing—enjoys a quadratic scaling enhancement over independent sensors due to spatial correlations of the drive field. For our DQS setup, we similarly witness a quadratic scaling over independent sensors as well as a constant factor improvement over the CL-DCS scheme, due to compounding the benefits of classical correlations from the drive field—which boosts the signal—and quantum correlations between the optical fields—which leads to a global, broadband reduction of the noise.

We stress that independent sensors cannot achieve the

performance of our proposed DQS array with the same amount of squeezing—no matter the input laser power. Moreover, if we allow for joint post-processing but do not allow for entanglement between the modes, then Mindependent squeezed vacua—each with N_s number of squeezed photons—must be utilized (see Appendix E 6 for further discussion) in order to achieve the same performance as our DQS setup. This implies that the improvement in our DQS scheme is not necessarily due to the amount of squeezed light that impinges on a single mechanical oscillator but, rather, is a consequence of the quantum correlations between the optical fields that impacts the mechanics as a collective.

We are furthermore able to show that our results are robust to small non-idealities, which we illustrate numerically with an example of two sensors in the inset of Fig. 3. We introduce inhomogeneity through the second sensor by taking $\gamma_2 = \gamma/(1-p)$, $\kappa_2 = \kappa/(1-p)$, and $G_2 = G_0(1-p)$ for the sensor's operating parameters, where $0 \le p \le 1$ and γ , κ , and G_0 are the mechanical damping rate, cavity damping rate, and vacuum optomechanical coupling rate of the first sensor, respectively (acquiring the same numerical values as in Fig. 2). Thus, for p > 0, the second sensor performs worse than the first sensor; nevertheless, a performance boost remains.

Discussions.— We have shown that an entangled optomechanical sensor array provides enhanced broadband sensitivity over classical setups. Our theory also applies to a recent proof-of-principle experiment [53] and futuristic optomechanical arrays, such as the proposed Windchime project for DM search [32, 33], and may be relevant to other array-based proposals [54, 55].

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The commutation relation of the creation and annihilation operator of the optical field in the Heisenberg picture is $[\hat{a}^{in}(t), \hat{a}^{in\dagger}(t')] = \delta(t - t')$. We define the quadratures of a mode as the real and imaginary parts of the annihilation operator \hat{a}^{in} , respectively, as $\hat{X}^{in}(t) = [\hat{a}^{in}(t) + \hat{a}^{in\dagger}(t)]/\sqrt{2}$, $\hat{Y}^{in}(t) = [\hat{a}^{in}(t) - \hat{a}^{in\dagger}(t)]/\sqrt{2}$. By taking Fourier transforms of the time domain fields, we define $\hat{X}^{in}(\omega) = [\hat{a}^{in}(\omega) + \hat{a}^{in\dagger}(-\omega)]/\sqrt{2}$, $\hat{Y}^{in}(\omega) = [\hat{a}^{in}(\omega) - \hat{a}^{in\dagger}(-\omega)]/\sqrt{2}$; where the annihilation operator $\hat{a}^{in}(\omega)$ in the frequency domain is Fourier transform of $\hat{a}^{in}(t)$ and obeys the commutation relation $[\hat{a}^{in}(\omega), \hat{a}^{in\dagger}(\omega')] = 2\pi\delta(\omega - \omega')$.

In optomechanics, the PSD of the quadratures \hat{X}^{in} , \hat{Y}^{in} are usually related to the back-action and shot noise. They can be calculated with Eq. (1). When the optical field is coherent, the PSD of the vacuum fluctuations are

$$S_{\hat{X}^{\text{in}}}^{\text{vac}} = \int d\omega' \frac{1}{2\pi} \left\langle \hat{X}^{\text{in}\dagger}(-\omega)\hat{X}^{\text{in}}(\omega') \right\rangle$$

$$= \int d\omega' \frac{1}{4\pi} \left\langle \hat{a}^{\text{in}}(-\omega)\hat{a}^{\text{in}\dagger}(\omega') \right\rangle$$

$$= \int d\omega' \delta(\omega - \omega') = \frac{1}{2}$$

$$S_{\hat{Y}^{\text{in}}}^{\text{vac}} = \int d\omega' \frac{1}{2\pi} \left\langle \hat{Y}^{\text{in}\dagger}(-\omega)\hat{Y}^{\text{in}}(\omega') \right\rangle$$

$$= \int d\omega' \frac{1}{4\pi} \left\langle \hat{a}^{\text{in}}(-\omega)\hat{a}^{\text{in}\dagger}(\omega') \right\rangle$$

$$= \int d\omega' \delta(\omega - \omega') = \frac{1}{2},$$

(A1)

When the optical field is in a two mode squeezed state (squeezed in the quadrature $\hat{X}^{\text{in}\prime}$; see below), the annihilation operator $\hat{a}^{\text{in}\prime}(\omega)$ is transformed by the linear unitary Bogoliubov transformation $\hat{a}^{\text{in}\prime}(\omega) = \sqrt{N_S + 1} \hat{a}^{\text{in}}(\omega) + \sqrt{N_S} \hat{a}^{\text{in}\dagger}(-\omega)$ and the quadrature operators $\hat{X}^{\text{in}\prime}(\omega)$, $\hat{Y}^{\text{in}\prime}(\omega)$ by the transformations $\hat{X}^{\text{in}\prime}(\omega) = (\sqrt{N_S + 1} + \sqrt{N_S})\hat{X}^{\text{in}}(\omega)$, $\hat{Y}^{\text{in}\prime}(\omega) = \hat{Y}^{\text{in}}(\omega)/(\sqrt{N_S + 1} + \sqrt{N_S})$. The symmetrized PSD of the quadratures are

$$\begin{split} \bar{S}_{\hat{X}^{\text{in}\prime}}^{\text{sqz}} &= (\sqrt{N_S + 1} + \sqrt{N_S})^2 \bar{S}_{\hat{X}^{\text{in}}} \\ &= \frac{(\sqrt{N_S + 1} + \sqrt{N_S})^2}{2}, \\ \bar{S}_{\hat{Y}^{\text{in}\prime}}^{\text{sqz}} &= \frac{\bar{S}_{\hat{Y}^{\text{in}}}}{(\sqrt{N_S + 1} + \sqrt{N_S})^2} \\ &= \frac{1}{2(\sqrt{N_S + 1} + \sqrt{N_S})^2}. \end{split}$$
(A2)

Appendix B: Aside remarks on DM search

1. SNR and long observing runs

In a DM search, primary objectives are to either measure the DM driving-force—therefore confirming the existence of DM—or (more likely) to exclude regions in physical parameter space (e.g., DM mass and coupling) where no such signal is found. This must be accomplished in the presence of various noise sources, whilst optimally utilizing resources (e.g., laser power, squeezing, etc.).

For a single detection interval (of size $\sim 1/\Delta_a$), the signal PSD of the DM-induced force is $\bar{S}_{F_{\rm dr}}(\omega) \sim F_{\rm DM}^2/\Delta_a$, where $F_{\rm DM}$ is the amplitude of the (partially coherent) drive. For the optomechanical setup, the noise in the force estimation is given generally by Eq. (7) in the main text. For simplicity, we approximate the Lorentzian PSD [20] of DM induced force as a delta-like peak of width Δ_a , centered about the (unknown) mass of the DM particle.

Consider a long observation run of length T_O , such that $\Delta_a T_O \gg 1$, where T_O is the total observation time, and decompose the total observation time into smaller intervals, each of size $\sim 1/\Delta_a$. Statistically combining the results from each detection interval, we find the SNR over the entire observation,

$$SNR = \frac{\bar{S}_{F_{dr}}(\omega)}{\bar{S}_{F_{noise}}(\omega)} \sqrt{\Delta_a T_O},$$
 (B1)

where the factor $\sqrt{\Delta_a T_O}$ is due to the law-of-large numbers. For longer observation times, T_O , smaller DM couplings can be probed at each frequency, ω , thus providing a method to detect or exclude DM candidates.

a. Random force perspective

We now take a different viewpoint and consider long integration times, $T_{\text{int}}\Delta_a \gg 1$, such that the DM-induced force is incoherent and captured by the random force, $\xi_F(t)$, while the (partially coherent) deterministic drive $F_{\text{dr}}(t) = 0 \forall t$. In this case, we take the ratio of the signal PSD (induced by the random force) and the other noise terms to obtain the SNR directly,

$$SNR = \frac{S_{\xi_F}(\omega)}{S_{\text{noise}}(\omega)},$$
 (B2)

As before, we consider a long observation time, such that $T_O \gg T_{\rm int}$, and then break the entire observation run into a large number of repetitions, $T_O/T_{\rm int}$. For fixed T_O , increasing $T_{\rm int}$ will not change the single-shot SNR in Eq. (B2) but will otherwise reduce the number of repetitions, resulting in a decreased SNR over the observation time, T_{O} . To increase the number of repetitions (and thus increase the SNR over the entire observing run), we could decrease the integration time, however we cannot do so arbitrarily, as the DM signal has a finite linewidth. Therefore, the best strategy is to set $T_{\rm int} \approx 1/\Delta_a$, where the precision is just enough to resolve DM signal. At this point, one can multiply the single-shot SNR of Eq. (B2) by the factor $\sqrt{T_O/T_{\rm int}} \approx \sqrt{T_O \Delta_a}$ from repeated measurements. Identifying $S_{\xi_F}(\omega) = S_{F_{dr}}$, one then arrives at Eq. (B1). Hence, from a data processing perspective,

it is useful to think about the DM-induced force as a partially coherent drive, as discussed in the main text.

2. Comments on figures of merit in a DM search

Through exhaustively thorough analyses, Refs. [42, 43] established the integrated sensitivity as a primary figure of merit in a DM search. On the other hand, in original DM search proposals with microwave cavities [8–13, 16, 17], a physically motivated figure of merit is the actual time that it takes to scan frequency space in search for a DM signal (by tuning the resonance frequency of the cavity)—which is the "scan time" for the DM search.

In cavity setups, it can be shown that the scan rate (the inverse of the scan time) is proportional to the squared SNR, integrated over all resonance frequencies [15, 31, 56]. It turns out that, for a microwave cavity detector, the integrated sensitivity [defined in Eq. (9) of the main text] and the scan-rate are equivalent (up to a proportionality constant). This equivalence is due to the fact that the microwave cavity response (characterized by a Lorentzian profile) depends solely on the detuning from the resonance frequency of the cavity. This equivalence does not extend to optomechanical setups, due to the non-Lorentzian response of the mechanics. In our work, we follow the suggestions of Refs. [42, 43] and thus take the integrated sensitivity as the preferred performance metric. The integrated sensitivity furthermore has an interpretation as the total amount of information about the signal over the entire frequency spectrum, which is important in broadband sensing scenarios, making this figure of merit precise and more generally useful; see, e.g., Ref. [57] for its use in a different context for twolevel sensors.

The integrated sensitivity has also been considered in broadband detection of stochastic gravitational-wave backgrounds [58–60].

Appendix C: Single sensor analyses of the simplified model

In our simplified model, the optical field experience a phase shift caused by the motion of the mirror (as shown in Fig. 4). $\hat{E}^{\text{in}}(t)$ and $\hat{E}^{\text{out}}(t)$ are the input and output field operators at the equilibrium position (q = 0) of the mirror, respectively. There is a phase rotation $2k\hat{q}(t')$ between the input and output field caused by the propagation. Here k is the angular wavenumber of the field and $\hat{q}(t')$ is the position operator of the mirror at time t'. t' is the time the wave arrives the mirror before its return back to position q = 0 at time t. But because the motion speed of the mirror is much less than the light speed, we can treat $\hat{q}(t') \approx \hat{q}(t)$ approximately. The input-output relation of the field can be obtained as $\hat{E}^{\text{out}}(t) = e^{2ik\hat{q}(t)}\hat{E}^{\text{in}}(t) = e^{i\zeta\hat{q}(t)}\hat{E}^{\text{in}}(t)$, where



Figure 4. Schematic of the simplified model. \hat{E}^{in} and \hat{E}^{out} are the input and output field operators, respectively; m, Ω , and γ are are the effective mass, resonance frequency, and damping rate of the mechanical oscillator, respectively; and \hat{q} is the position operator of the mechanical oscillator.

 $\zeta = 2\Omega_L/c$. With the input field decomposed in the frequency domain, considering the loss, the output field is

$$\hat{E}^{\text{out}}(t) = e^{\mathrm{i}\zeta\hat{q}(t)}\hat{E}^{\text{in}}(t)$$

$$\approx e^{-\mathrm{i}\Omega_L t} \Big[\eta E_0 \left(1 + \mathrm{i}\zeta\hat{q}(t)\right) + \int \frac{\mathrm{d}\omega}{2\pi} \eta \hat{a}^{\text{in}}(\omega) e^{-\mathrm{i}\omega t} \Big] + \cdots, \quad (C1)$$

The output field is measured via homodyne detection with the local oscillator field to be $\hat{E}^{\text{in}}(t)i/E_0$. The detected quadrature is

$$\hat{Y}^{\text{out}}(t) = \hat{E}^{\text{out}}(t)[-\hat{E}^{\text{in}*}(t)i/E_0] + h.c.$$

$$= -i\Big[\eta E_0 \left(1 + i\zeta \hat{q}(t)\right) + \int \frac{d\omega}{2\pi} \eta \hat{a}^{\text{in}}(\omega)e^{-i\omega t} + \cdots\Big] + h.c.$$

$$= -i\Big[i2\eta E_0\zeta \hat{q}(t) + \int \frac{d\omega}{2\pi} \eta \hat{a}^{\text{in}}(\omega)e^{-i\omega t} - \int \frac{d\omega}{2\pi} \eta \hat{a}^{\text{in}\dagger}(\omega)e^{i\omega t} + \cdots\Big].$$
(C2)

 \hat{Y}^{out} in the frequency domain is obtained by Fourier transformation,

$$\hat{Y}^{\text{out}}(\omega') = \int \hat{Y}^{\text{out}}(t)e^{i\omega't}dt$$

$$= -i\eta[\hat{a}^{\text{in}}(\omega') - \hat{a}^{\text{in}\dagger}(-\omega')] + 2\eta E_0 \zeta \hat{q}(\omega') + \cdots$$

$$= \sqrt{2}\eta \Big[\hat{Y}^{\text{in}}(\omega') + \sqrt{2}E_0 \zeta \hat{q}(\omega') \Big] + \cdots$$
(C3)

The dynamics of the mechanical oscillator is determined by the radiation force, $\hat{F}_{\rm rad}$, from the light field probing the motion of the mechanics, the external driving force, $F_{\rm dr}$, exerted on the mechanics by the DM field and the random force, $\hat{\xi}_T$ induced by a thermal bath at temperature T. The motion equation of the mechanical

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oscillator is

$$m\ddot{\hat{q}} + 2m\gamma\dot{\hat{q}} + k\hat{q} = F_{\rm dr}(t) + \hat{F}_{\rm rad}(t) + \sqrt{4m\gamma K_B T} \hat{\xi}_T(t).$$
(C4)

The oscillator position \hat{q} can be obtained by solving the Langevin equation in the Fourier domain, resulting in the solution,

$$\hat{q}(\omega) = \frac{\chi_{\omega}}{m\Omega} \Big[F_{\rm dr}(\omega) + \hat{F}_{\rm rad}(\omega) + \sqrt{4m\gamma K_B T} \hat{\xi}_T(\omega) \Big],$$
(C5)

with m, γ , and Ω being the oscillator mass, mechanical damping rate, and mechanical resonance frequency, respectively. The random force $\hat{\xi}_T(\omega)$ has zero-mean and is Gaussian distributed, with $\langle \hat{\xi}_T(\omega) \hat{\xi}_T(\omega') \rangle = 2\pi \delta(\omega - \omega')$, where K_B is the Boltzmann constant. Finally, $\chi_{\omega} = \Omega/(\Omega^2 - \omega^2 - i2\gamma\omega)$ is the complex linear response of the mechanics.

The radiation pressure force in time domain is given by

$$\hat{F}_{\rm rad}(t) = \kappa \hat{E}^{\rm in\,\dagger}(t) \hat{E}^{\rm in}(t) = \kappa E_0 \int \frac{\mathrm{d}\omega}{2\pi} \hat{a}^{\rm in}(\omega) e^{-\mathrm{i}\omega t} + h.c., \quad (C6)$$

where $\kappa = 2\hbar\Omega_L/c$ is the momentum change of a photon reflected from the mirror. Its spectral amplitude is

$$\hat{F}_{\rm rad}(\omega') = \int \hat{F}_{\rm rad}(t) e^{i\omega' t} dt = \kappa E_0 [\hat{a}^{\rm in}(\omega') + \hat{a}^{\rm in\dagger}(-\omega')]$$
$$= \sqrt{2}\kappa E_0 \hat{X}^{\rm in}(\omega').$$
(C7)

Substituting Eq. (C5) and Eq. (C7) to Eq. (C3), we obtain

$$\hat{Y}^{\text{out}}(\omega) = \sqrt{2}\eta \left\{ \hat{Y}^{\text{in}}(\omega) + \frac{\sqrt{2}E_0\zeta\chi_\omega}{m\Omega} \left[\sqrt{2}\kappa E_0 \hat{X}^{\text{in}}(\omega) + F_{\text{dr}}(\omega) + \sqrt{4m\gamma K_B T} \hat{\xi}_T(\omega) \right] \right\} + \cdots . \quad (C8)$$

The force estimator can be then expressed with the phase quadrature as

$$\hat{F}(\omega) = \frac{m\Omega}{2\eta E_0 \zeta \chi_\omega} \hat{Y}^{\text{out}}(\omega)$$

= $B(\omega) \hat{Y}^{\text{in}}(\omega) + \sqrt{2}\kappa E_0 \hat{X}^{\text{in}}(\omega) + F_{\text{dr}}(\omega)$
+ $\sqrt{4m\gamma K_B T} \hat{\xi}_T(\omega) \Big] + \cdots$ (C9)

where

$$B(\omega) = \frac{m\Omega}{\sqrt{2}E_0\zeta\chi_\omega}.$$
 (C10)

from which the PSD of the force can be derived,

$$\begin{split} S_{\hat{F}}(\omega) &= |B(\omega)|^2 \Big[S_{\hat{Y}^{\text{in}}}(\omega) + \frac{1 - \eta^2}{2\eta^2} \Big] + 2\kappa^2 E_0^2 S_{\hat{X}^{\text{in}}}(\omega) \\ &+ \frac{1}{2\pi} \int d\omega' \left\langle B^*(-\omega) \hat{Y}^{\text{in}\dagger}(-\omega) \sqrt{2}\kappa E_0 \hat{X}^{\text{in}}(\omega') \right\rangle \\ &+ \frac{1}{2\pi} \int d\omega' \left\langle \sqrt{2}\kappa E_0 \hat{X}^{\text{in}\dagger}(-\omega) B(\omega') \hat{Y}^{\text{in}}(\omega') \right\rangle \\ &+ 4m\gamma K_B T + S_{F_{\text{dr}}}(\omega) \\ &= |B(\omega)|^2 \Big[S_{\hat{Y}^{\text{in}}}(\omega) + \frac{1 - \eta^2}{2\eta^2} \Big] + 2\kappa^2 E_0^2 S_{\hat{X}^{\text{in}}}(\omega) \\ &+ B'^*(-\omega) S_{\hat{Y}^{\text{in}}\hat{X}^{\text{in}}}(\omega) + B'(-\omega) S_{\hat{X}^{\text{in}}\hat{Y}^{\text{in}}}(\omega) \\ &+ 4m\gamma K_B T + S_{F_{\text{dr}}}(\omega), \end{split}$$
(C11)

where

$$B'(\omega) = \sqrt{2\kappa E_0} B(\omega). \tag{C12}$$

In the derivation above, we have used the stationary statistic properties $\langle \hat{O}^{\dagger}(\omega)\hat{O}'(\omega')\rangle = f_{\hat{O}\hat{O}'}(\omega)\delta(\omega - \omega')$ for \hat{O} and \hat{O}' to be anyone of \hat{X}^{in} and \hat{Y}^{in} . There is no correlations between the field quadrature, F_{dr} and $\hat{\xi}_T$, so the PSD of the force estimator is just the summation of the PSD of each of them. The PSD of force noise is just the summation of the residual terms apart from $S_{F_{\text{dr}}}(\omega)$ on the right hand side of Eq. (C11). Therefore the symmetrized PSD of noise is

$$\begin{split} \bar{S}_{F_{\text{noise}}}(\omega) &= 4m\gamma K_B T + |B(\omega)|^2 \Big[\bar{S}_{\hat{Y}^{\text{in}}}(\omega) + \frac{1 - \eta^2}{2\eta^2} \Big] \\ &+ 2\kappa^2 E_0^2 \bar{S}_{\hat{X}^{\text{in}}}(\omega) + \frac{B'^*(-\omega)S_{\hat{Y}^{\text{in}}\hat{X}^{\text{in}}}(\omega)}{2} \\ &+ \frac{B'(-\omega)\left[S_{\hat{Y}^{\text{in}}\hat{X}^{\text{in}}}(-\omega) + S_{\hat{X}^{\text{in}}\hat{Y}^{\text{in}}}(\omega)\right]}{2} \\ &+ \frac{B'(\omega)S_{\hat{X}^{\text{in}}\hat{Y}^{\text{in}}}(-\omega)}{2} \\ &= 4m\gamma K_B T + |B(\omega)|^2 \Big[\bar{S}_{\hat{Y}^{\text{in}}}(\omega) + \frac{1 - \eta^2}{2\eta^2} \Big] \\ &+ 2\kappa^2 E_0^2 \bar{S}_{\hat{X}^{\text{in}}}(\omega) + 2\text{Re}[B'(-\omega)\tilde{S}_{\hat{X}^{\text{in}}\hat{Y}^{\text{in}}}(\omega)], \quad (C13) \end{split}$$

where we have used the properties $B'(\omega) = B'^*(-\omega)$, $S_{\hat{O}\hat{O}'}(\omega) = S^*_{\hat{O}'\hat{O}}(\omega)$ in the derivation above.

Appendix D: Single cavity optomechanics

1. Brief review of dynamics

We start with the non-linear Hamiltonian of a cavity with one oscillating mirror. Let \hat{X} and \hat{Y} be the amplitude and phase quadratures, respectively, of the intracavity field and \hat{Q} and \hat{P} be the (dimensionless) position and momentum operators, respectively, of the movable mirror, such that $[\hat{X}, \hat{Y}] = [\hat{Q}, \hat{P}] = i$, with all other commutators vanishing. The Hamiltonian is then given by a sum of terms,

$$\hat{H} = \hat{H}^{\text{free}} + \hat{H}^{\text{int}} + \hat{H}^{\text{dr}}, \qquad (D1)$$

where,

$$\hat{H}^{\text{free}} = \frac{\hbar\omega_c}{2} \left(\hat{X}^2 + \hat{Y}^2 \right) + \frac{\hbar\Omega}{2} \left(\hat{Q}^2 + \hat{P}^2 \right) \qquad (\text{D2})$$

$$\hat{H}^{\text{int}} = 2\hbar G_0 \hat{Q} \left(\hat{X}^2 + \hat{Y}^2 \right) \tag{D3}$$

$$\hat{H}^{\rm dr} = \hbar \mathcal{E} \hat{X}.\tag{D4}$$

The first term is the free evolution of the intra-cavity quadrature operators (rotating at the cavity resonance-frequency, $\Omega_c = \Omega_L$, which we assume is on-resonance with the laser drive) plus the free-motion of the mirror (oscillating at the mechanical frequency, Ω), the second term is the radiation-pressure interaction, which induces a shift of the cavity resonance-frequency depending on the mirror's position, and the last term is a linear-drive of the cavity field, which can be related to the input laser drive flux E_0^2 (see Ref. [39] for explicit expressions). Here, G_0 is the (normalized) vacuum optomechanical coupling rate. For a Fabry-Perot cavity of length L, $G_0 = \frac{\Omega_c}{L} \sqrt{\frac{\hbar}{2m\Omega}}$, where m is the mass of the oscillator.

One can linearise the Hamiltonian by expanding around the steady-state values of the intra-cavity field and the mirror's motion (the latter being induced by a strong field inside the cavity). We then go to the rotating frame of the cavity. This leads to the linearapproximation of the Hamiltonian in the rotating frame of the laser,

$$\hat{H} = \frac{\hbar\Omega}{2} \left(\hat{Q}^2 + \hat{P}^2 \right) + 2\hbar G \hat{Q} \hat{X}, \qquad (D5)$$

where $G \equiv EG_0$ is the cavity-enhanced optomechanical coupling-rate and E is the intra-cavity amplitude (taken to be real). The intra-cavity amplitude is related to the input flux of the laser-drive, E_0 , via,

$$E^2 = \frac{4\kappa_r}{\kappa^2} E_0^2,\tag{D6}$$

where κ is the total dissipation-rate of the cavity and κ_r is the dissipation-rate to the readout port. If there is loss at a rate κ_ℓ , then $\kappa = \kappa_r + \kappa_\ell$. Note that, given the laser field has frequency Ω_L , the input laser-power is $P_{\rm in} = \hbar \Omega_L E_0^2 \approx \hbar \Omega_L \kappa E^2/4$, where the approximation assumes over-coupling ($\kappa_r \approx \kappa$).

We include noise by assuming that the intra-cavity field and the mechanical motion of the mirror are dissipatively coupled to bath-modes— $(\hat{X}^{\text{in}}, \hat{Y}^{\text{in}})$ and $(\hat{Q}^{\text{in}}, \hat{P}^{\text{in}})$ —at rates κ and γ , respectively, such that $[\hat{X}_{\text{in}}(t), \hat{Y}_{\text{in}}(t')] =$ $[\hat{Q}_{\text{in}}(t), \hat{P}_{\text{in}}(t')] = i\delta(t - t')$. In words, these modes represent an incoming photon-flux for the cavity field and an incoming phonon-flux for the oscillator's motion. The interactions with the baths lead to a coupled set of firstorder, linear differential equations,

$$\frac{\mathrm{d}\hat{X}}{\mathrm{d}t} = -\frac{\kappa}{2}\hat{X} + \sqrt{\kappa}\hat{X}_{\mathrm{in}},\tag{D7}$$

$$\frac{\mathrm{d}Y}{\mathrm{d}t} = \frac{\kappa}{2}\hat{Y} + \sqrt{\kappa}\hat{Y}_{\mathrm{in}} - 2G\hat{Q},\tag{D8}$$

$$\frac{\mathrm{d}\hat{Q}}{\mathrm{d}t} = \Omega\hat{P},\tag{D9}$$

$$\frac{\mathrm{d}\hat{P}}{\mathrm{d}t} = -\Omega\hat{Q} - 2\gamma\hat{P} + 2\sqrt{\gamma}\hat{P}_{\mathrm{in}} - 2G\hat{X}.$$
 (D10)

The outgoing fluxes can be found from the time-reversal of Eqs. (D7)-(D10) and are related to the ingoing flux via the input-output relations $\hat{X}^{\text{out}} = \hat{X}^{\text{in}} - \sqrt{\kappa}\hat{X}$ etc., from which the (spectral) output amplitude and phase quadratures can be found,

$$\hat{X}^{\text{out}}(\omega) = -\mathrm{e}^{\mathrm{i}\varphi_{\omega}}\hat{X}^{\text{in}}(\omega), \qquad (D11)$$

$$\hat{Y}^{\text{out}}(\omega) = -e^{i\varphi_{\omega}}\hat{Y}^{\text{in}}(\omega) + 2\sqrt{2\gamma C_{\omega}}\hat{Q}(\omega), \qquad (D12)$$

with the latter expression agreeing with Eq. (3) of the main text and

$$\hat{Q}(\omega) = 2\sqrt{\gamma}\chi_{\omega} \left(\hat{P}^{\mathrm{in}}(\omega) - \sqrt{2C_{\omega}}\hat{X}^{\mathrm{in}}(\omega)\right) + Q_{\mathrm{dr}}(\omega),$$
(D13)

where we have added an additional drive term $Q_{\rm dr}(\omega)$ due to a background, classical force.

2. Bad cavity limit

Here we show that, in the bad cavity limit, the simplified model (Appendix C) and the cavity optomechanics model agree. The goal is to show that the overall inputoutput relations agree. We accomplish this by comparing the force estimator of the simplified model, Eq. (C9), to the force estimator of the optomechanical model, Eq. (6). Writing the latter out explicitly,

$$\hat{F}(\omega) = \frac{\mathrm{e}^{-\mathrm{i}\varphi_{\omega}/2}}{\chi_{\omega}} \sqrt{\frac{\hbar m\Omega}{8\gamma |C_{\omega}|}} \hat{Y}^{\mathrm{out}}(\omega)$$

$$= -\frac{1}{\chi_{\omega}} \sqrt{\frac{\hbar m\Omega}{8\gamma |C_{\omega}|}} \hat{Y}^{\mathrm{in}}(\omega) + \sqrt{4\hbar m\gamma \Omega} \hat{P}^{\mathrm{in}}(\omega)$$

$$+ F_{\mathrm{dr}} - \sqrt{8\hbar m\gamma \Omega C_{\omega}} \hat{X}^{\mathrm{in}}(\omega). \quad (D14)$$

In the bad cavity limit, $e^{i\varphi_{\omega}} \approx 1$ and $C_{\omega} \approx 2G^2/\gamma \kappa = 8E_0^2 G_0^2/\gamma \kappa^2$. If we shift the phase reference of the homodyne detection by π , we have $\hat{Y}^{\prime in}(\omega) = -\hat{Y}^{in}(\omega)$ and $\hat{X}^{\prime in}(\omega) = -\hat{X}^{in}(\omega)$, and Eq. (D14) can be expressed as

$$\hat{F}(\omega) = \frac{\sqrt{\hbar m \Omega \kappa}}{8E_0 G_0 \chi_\omega} \hat{Y}^{\prime \text{in}}(\omega) + \sqrt{4\hbar m \gamma \Omega} \hat{P}^{\text{in}}(\omega) + F_{\text{dr}} + \frac{8E_0 G_0}{\kappa} \sqrt{\hbar m \Omega} \hat{X}^{\prime \text{in}}(\omega), \quad (D15)$$

Comparing Eq.(C9) and Eq.(D15) (at unity efficiency $\eta = 1$), we make the following correspondence

$$\frac{\sqrt{\hbar m \Omega} \kappa}{8 E_0 G_0 \chi_\omega} = B(\omega), \tag{D16}$$

where $B(\omega)$ can be found in Eq. (C10). This implies that

$$\hbar \zeta = \frac{4G_0}{\kappa} \sqrt{2\hbar m \Omega} = \frac{4\hbar \Omega_L}{L\kappa}.$$
 (D17)

where we have used $E^2 = (4/\kappa)E_0^2$ and taken $G_0 \sim \sqrt{\hbar/2m\Omega}\Omega_L/L$, which holds identically for a Fabry-Perot cavity. In the bad cavity limit of the cavity optomechanics model, the output mirror of the cavity is seen as a window (i.e., almost completely transparent) and thus $\kappa \sim c/L$, which is due to the fact that it roughly takes a time L/c for a photon to leave the cavity after entry through the mirror. Consequently, in this limit, the PSD expressions for each model agree.

3. Squeezing enhancement

We introduce squeezing into the input optical field, which generally correlates the phase and amplitude quadratures and allows us to go below the SQL when interrogating the mechanics. Given a squeezing strength, r, and a squeezing angle, θ , the PSDs of the input optical field are,

$$\bar{S}_{\hat{Y}^{\text{in}}\hat{Y}^{\text{in}}}(\omega) = \frac{1}{2} \left(e^{-2r} \cos^2 \theta + e^{2r} \sin^2 \theta \right), \qquad (D18)$$

$$\bar{S}_{\hat{X}^{\text{in}}\hat{X}^{\text{in}}}(\omega) = \frac{1}{2} \left(e^{2r} \cos^2 \theta + e^{-2r} \sin^2 \theta \right), \qquad (D19)$$

$$\tilde{S}_{\hat{X}^{\mathrm{in}}\hat{Y}^{\mathrm{in}}}(\omega) = \tilde{S}_{\hat{Y}^{\mathrm{in}}\hat{X}^{\mathrm{in}}}(\omega) = \frac{1}{2}\cos\theta\sin\theta(\mathrm{e}^{2r} - \mathrm{e}^{-2r}).$$
(D20)

We then assume the mechanical noise in the system is approximately flat, such that $\bar{S}_{P^{\text{in}}P^{\text{in}}}(\omega) = K_B T/\hbar\Omega$. Substituting these expressions into Eq. (7) of the main text and rearranging, we obtain,

$$\bar{S}_{F_{\text{noise}}}^{\text{sqz}}(\omega) = \frac{\hbar m \Omega}{16\gamma |C_{\omega}| |\chi_{\omega}|^2} \left(\left| \cos \theta - 8\gamma |C_{\omega}| \chi_{\omega} \sin \theta \right|^2 e^{-2r} + \left| \sin \theta + 8\gamma |C_{\omega}| \chi_{\omega} \cos \theta \right|^2 e^{2r} \right) + 4m\gamma K_B T, \quad (D21)$$

where we have used $\chi_{\omega}^* = \chi_{-\omega}$. In terms of the number of squeezed photons, N_s , we have $e^{-2r} = 1/(\sqrt{N_s} + \sqrt{N_s + 1})^2$.

For a fixed amount of squeezed photons, $N_s > 1$, and at a given frequency $\omega \neq \Omega$, there exists a laser-drive amplitude, E_{ω}^{\star} , and a squeezing angle, θ_{ω}^{\star} , such that the force noise dips below the SQL. This can be seen by comparing the (dashed) black and red curves in Fig. 5, which are the ultimate performances for classical and squeezingenhanced detection, respectively, for some finite temperature, T (T = 10K here).

In practice, the SQL and the squeezed-noise limit can only be reached at a particular frequency because the laser power, E, is constant (independent of ω). Since we are concerned with broadband detection performance, we can otherwise choose the laser power to optimize the integrated sensitivity of the detector, such that $E = \tilde{E}^*$, in accordance with the optimization schemes discussed in the main text. This leads, for instance, to a practical classical-limit in broadband detection sensitivity without squeezing; see the dark blue curve [Practical CL ($E = \tilde{E}^*$)] in Fig. 5.

Squeezing can be at classical limits in broadband detection if the squeezing angle, $\theta,$ is appropriately chosen, a known result [46, 47, 49–52]. [Note that enhanced broadband sensing is best highlighted by the integrated sensitivity, which we plot in Fig. 2 of the main text. We give qualitative analysis by directly observing the noise spectrum here.] This is especially pronounced if the squeezing angle is frequency tunable, such that we may operate the system at the optimal point, $\theta = \theta_{\omega}^{\star}$. We can qualitatively observe this by inspecting the squeezed-noise in Fig. 5 [light blue curve; Squeezed $(E = \widetilde{E}^{\star}, \theta = \theta_{\omega}^{\star})$] and comparing it to the classical limit (dark blue curve). There, we notice that squeezing admits lower noise than the practical classical-limit at all frequencies and even surpasses the SQL+Thermal noise limit [black dashed curved] for a vast range of frequencies. Similar (though not as advantageous) benefits arise when the squeezing angle is not arbitrarily tunable but may be nonetheless be chosen to take on a single value which maximizes the integrated sensitivity [orange curve; Squeezed $(E = \widetilde{E}^{\star}, \theta = \widetilde{\theta}^{\star})].$

We now consider the effect of loss on performance. As mentioned in the main text, for detection loss $1 - \eta^2$, the force noise when loss is present can be found by the simple substitution, $\bar{S}_{F_{\text{noise}}} \rightarrow \bar{S}_{F_{\text{noise}}} + \frac{1-\eta^2}{\eta^2} (\hbar m \Omega / 16 \gamma |C_{\omega}| |\chi_{\omega}|^2)$. [We have



Figure 5. Force noise (in units Newtons/ $\sqrt{\text{Hz}}$) for a single optomechanical sensor. The amount of squeezing for each squeezed curve is taken as 4.2 dB (equivalent $N_s = .25$ squeezed photons). Operational parameters are $\Omega = 2\pi \times 1 \text{ kHz}$, T = 10 K, $Q \equiv \Omega/2\gamma = 10^9$ (thus, $2\gamma = 2\pi \times 1 \mu \text{ Hz}$), $\kappa = 2\pi \times 16 \text{ MHz}$, m = 10 ng, and $G_0 = 2\pi \times 121 \text{ Hz}$ (cf. Refs. [20, 45]). Values \tilde{E}^* and $\tilde{\theta}^*$ are chosen via $(\tilde{E}^*, \tilde{\theta}^*) = \arg \max[\mathcal{I}_{\Omega}]$, whereas E^{ω}_{ω} and θ^{\star}_{ω} are chosen via $(E^{\omega}_{\omega}, \theta^{\star}_{\omega}) = \arg \max[\tilde{S}_{\hat{F}_{\text{dr}}}/\bar{S}_{F_{\text{noise}}}]$.

thus factored out loss from the signal and pushed it in the noise, hence the η^2 in the denominator.] The first term is the force noise without loss, and the second term is additional measurement noise from the vacuum fluctuations of the loss port. As an example, for the SQL, $\bar{S}_{F_{\text{noise}}}^{\text{SQL}} \rightarrow \bar{S}_{F_{\text{noise}}}^{\text{SQL}}/\eta$; this can be shown by optimizing the force noise (assuming vacuum fluctuations for the input optical field), from which one finds the optimal cooperativity, $|C_{\omega}| = 1/(8\eta\gamma|\chi_{\omega}|)$, that leads to the aforementioned expression. In Fig. 6, we plot the integrated sensitivity for a mechanical oscillator with resonance frequency $\Omega = 1 \,\mathrm{kHz}$ versus loss, for various detection configurations (with and without squeezing). For $1 - \eta^2 > 0$, the benefits from squeezing is ultimately limited by loss (as well as the thermal floor). Furthermore, the overall performance of each configuration degrades as the amount of loss increases, however squeezing is still beneficial for all non-zero values of the loss.

Appendix E: Optmechanical array analysis

Below are detailed analyses of an array of optomechanical sensors, where we discuss some subtleties about the interaction Hamiltonian (describing the coupling between the mechanics and radiation) for an array, provide derivations of the force noise for an array of optomechanical sensors, discuss the SQL of the array, derive an explicit expression for the noise when input squeezed radiation is



Figure 6. Integrated sensitivity versus loss, $1 - \eta^2$. Mechanical resonance frequency is taken as $\Omega = 1$ kHz; all other operational parameters are the same as in Fig. 5.

utilized, and, finally, derive an optimal squeezing angle for the array.

1. Optomechanical interaction Hamiltonian for an array

Consider the interaction Hamiltonian for the jth sensor [see, e.g., Eq. (D3)] in an optomechanical quantum array,

$$\hat{H}_{j}^{\text{int}} = 4\hbar g_{0;j} \hat{Q}_{j} \left(\hat{a}_{j}^{\dagger} \hat{a}_{j} \right), \qquad (\text{E1})$$

where we have rewritten the expression in terms of the intra-cavity annihilation and creation operators, \hat{a}_j and \hat{a}_j^{\dagger} . The annihilation operator in the Heisenberg picture is given in the Fourier domain by,

$$\hat{a}_j = \int \mathrm{d}\omega \hat{a}_j(\omega) \mathrm{e}^{-\mathrm{i}\omega t}.$$
 (E2)

We now linearise the interaction Hamiltonian by considering the intra-cavity field around its mean spectral amplitude, $\beta_j(\omega)$. This amplitude is linearly related to the input field amplitude, $\beta_{in;j}(\omega)$, via

$$\beta_j(\omega) = \frac{\sqrt{\kappa_{r;j}}}{\mathrm{i}(\omega - \Omega_c) + \kappa_j/2} \beta_j^{\mathrm{in}}(\omega), \qquad (E3)$$

where Ω_C is the resonant frequency of the cavity, κ_j is the total dissipation rate of the cavity, and $\kappa_{r;j}$ is the dissipation rate to the readout port. For simplicity, we assume all cavities in the array have approximately the same resonance frequency. Let $E_0(\omega)$ be the input laserfield amplitude to the primary mode (i.e., the 0th input mode; all other modes are vacuum) prior to the beamsplitter array. Then, $\beta_j^{\text{in}}(\omega) = w_{j0}(\omega)E_0(\omega)$ and

$$\beta_j(\omega) = w_{j0}(\omega) \left(\frac{\sqrt{\kappa_{r;j}}}{i(\omega - \Omega_c) + \gamma_j/2} E_0(\omega) \right)$$

$$\equiv w_{j0}(\omega) E_j(\omega),$$
(E4)

where $E_j(\omega)$ is the intra-cavity amplitude if the entire laser field impinged on the cavity (i.e., in the absence of the power-dividing array). We now make the substitution $\hat{a}_j(\omega) \longrightarrow \hat{a}_j(\omega) + \beta_j(\omega)$ into Eq. (E2), using the equations above, and assuming the input field, $E_0(\omega)$, to be sharply peaked around the laser frequency, Ω_L . We then find,

$$\hat{a}_{j} \to \hat{a}_{j} + \int d\omega \, w_{j0}(\omega) E_{j}(\omega) e^{-i\omega t}$$

$$\approx \hat{a}_{j} + w_{j0}(\Omega_{L}) E_{j}(\Omega_{L}) e^{-i\Omega_{C} t}.$$
(E5)

We shall further assume that the laser-frequency is onresonance with the cavities, such that $\Omega_L = \Omega_C$. Then, substituting the prior expression back into the interaction Hamiltonian of Eq. (E1) and linearizing the result, we obtain, in the rotating reference frame of the cavity/laser,

$$\begin{aligned} \hat{H}_{j}^{\text{int}} &\approx 4\hbar g_{0;j} \hat{Q}_{j} |w_{j0}(\Omega_{L}) E_{j}| \left(\hat{a}_{j} \mathrm{e}^{-\mathrm{i}\Phi_{j}} + \hat{a}_{j}^{\dagger} \mathrm{e}^{\mathrm{i}\Phi_{j}} \right) \\ &\equiv 2\hbar g_{j} \hat{Q}_{j} \hat{X}_{j}(\Phi_{j}), \end{aligned} \tag{E6}$$

where $\Phi_j \equiv \arg[E_0] + \arg[w_{j0}(\Omega_L)]; \quad g_j \equiv g_{0;j}|w_{j0}(\Omega_L)E_j|; \quad \hat{X}_j(\Phi_j) \equiv \operatorname{Re}\{\hat{a}_j e^{-i\Phi_j}\};$ and we have discarded a term $\propto \hat{Q}_j|w_{j0}(\Omega_L)E_j|^2$ which simply determines the steady-state of the mechanical oscillations.

In gist, these results show that:

- The bare coupling parameter of the *j*th sensor, $g_{0;j}$, gets enhanced to g_j , but only by a fraction of the total input laser field. Thus, if $C_j(\omega)$ denotes the mechanical cooperativity (as defined per Eq. (4)) of the *j*th sensor when the *total* laser field, E_0 , interacts with the mechanics, then the actual cooperativity of the mechanics is $C'_j(\omega) = |w_{j0}(\Omega_L)|^2 C_j(\omega)$, as claimed in the main text.
- From $\hat{X}_j(\Phi_j)$ and the definition of Φ_j , we observe that the beam-splitter array can cause the quadrature bases of sensors within the array to differ, due to the angle $\arg[w_{j0}(\Omega_L)]$. This can cause problems when power-combining signals later on, as the signal at the *j*th sensor is encoded in the *j*th phase quadrature, $\hat{Y}_j(\Phi_j)$, and we want to align these quadratures in order to maximize signal output. To avoid potential mishaps here, we can choose $\arg[w_{j0}(\Omega_L)] = 0 \forall j$.

2. Input-output relations

We now derive the input output relations for the sensor array depicted in Fig. 1, which leads to the general force noise in Eq. (E24) of the main text. A non-trivial input mode, \hat{Y}_0^{in} , is mixed with vacuum fluctuations from M-1idling input modes via a beam-splitter array governed by the weights $\{w_{k0}\}$, with $w_{k0} \in \mathbb{C}$. Just after the beamsplitter array, the phase-quadrature that impinges on the the *n*th mechanical-sensor is given as,

$$\hat{Y}_{n}^{\prime \text{ in}} = w_{n0}\hat{Y}_{0}^{\text{in}} + \text{vac},$$
 (E7)

where "vac" indicates the (linear combination of) vacuum fluctuations from the M-1 idle input modes. After interaction with the mechanics, the phase quadrature at the output of the *n*th cavity is, which we measure via homodyne detection,

$$\hat{Y}_{n}^{\text{out}} = -\mathrm{e}^{\mathrm{i}\varphi_{n}(\omega)}\hat{Y}_{n}^{\prime \text{ in}} + 2\sqrt{2\gamma_{n}C_{n}^{\prime}(\omega)}\hat{Q}_{n} - 8\gamma_{n}C_{n}^{\prime}(\omega)\chi_{n}(\omega)\hat{X}_{n}^{\prime \text{ in}}, \quad (E8)$$

where \hat{Q}_n is defined similarly as in Eq. (5), \hat{X}'_n in is defined likewise to Eq. (E7) (which is required to preserve the canonical commutation relations between the phase and amplitude quadratures), and $C'_n(\omega) \equiv$ $C_n(\omega)|w_{n0}(\Omega_L)|^2$, with Ω_L the laser frequency, which we have assumed to be the same for all sensors and assumed to be resonant with each cavity. Here, the cooperativity $C_n(\omega)$ [see Eq. (4) for an explicit expression] is defined with respect to the total power E_0^2 . In our work, we take $\arg[w_{n0}(\Omega_L)] = 0$ (or some constant independent of n). Without this choice, the quadrature bases of the output radiation at each sensor will not be aligned, which will lead to poorer performance when attempting to combine the signal amplitudes; see previous section for further details about this.

After detection, the quadrature measurements can be converted to force measurements via the conversion formula (6), from which the signal and noise PSDs can be derived.

3. Noise Analysis

From the previous section and Eq. (10) of the main text, it follows that the PSD noise is,

$$S_{\text{noise}}(\omega) = \sum_{j,k} W_{0j}^* W_{0k} \left\langle \hat{F}_j^{\dagger} \hat{F}_k \right\rangle_{\text{noise}}, \qquad (E9)$$

where we have dropped frequency dependence for brevity. The subscript "noise" here indicates that we disregard the signal contribution to the PSD when evaluating this expression. The expression $\langle \hat{F}_j^{\dagger} \hat{F}_k \rangle_{\text{noise}}$ has four main terms: mechanical thermal noise, shot noise, back-action, and quadrature correlations. We consider the noise contributions in parts.

Mechanical thermal fluctuations at each sensor con-

tribute to the total noise and obey the relations,

$$\left\langle \hat{F}_{j}^{\dagger} \hat{F}_{k} \right\rangle_{\Theta} \equiv \frac{\sqrt{\hbar^{2} m_{k} m_{j} \Omega_{k} \Omega_{j}}}{\chi_{k} \chi_{j}^{*}} \left\langle \hat{Q}_{k}^{\dagger} \hat{Q}_{j} \right\rangle_{\text{noise}}$$
(E10)
$$= 4 \sqrt{\hbar^{2} m_{k} m_{j} \Omega_{k} \Omega_{j} \gamma_{k} \gamma_{j}} \underbrace{\left\langle \hat{P}_{j}^{\text{in} \dagger} \hat{P}_{k}^{\text{in}} \right\rangle}_{=\delta_{jk} (K_{B} T_{k} / \hbar \Omega_{k})}$$
(E11)
$$= \delta_{jk} \left(4 m_{k} \kappa_{k} K_{B} T_{k} \right),$$
(E12)

where Θ labels "thermal" and Eq. (5) was used to move from the first line to the second. The total thermal contribution to the PSD of Eq. (E9) is then,

$$S_{\text{noise}}^{\Theta} \equiv \sum_{k=0}^{M-1} |W_{0k}|^2 \left(4m_k \gamma_k K_B T_k\right), \quad (E13)$$

which is just an average thermal noise over all sensors, according to the probability distribution $|W_{0k}|^2$. [Recall, $0 \le |W_{0k}|^2 \le 1$ and $\sum_k |W_{0k}|^2 = 1$.]

The fluctuations in the phase quadrature leads to shotnoise (SN), which is determined by,

$$\left\langle \hat{F}_{j}^{\dagger}\hat{F}_{k}\right\rangle_{\rm SN} \equiv \frac{\mathrm{e}^{\mathrm{i}(\varphi_{k}-\varphi_{j})/2}}{8\chi_{k}\chi_{j}^{*}}\sqrt{\frac{\hbar^{2}m_{k}m_{j}\Omega_{k}\Omega_{j}}{\gamma_{k}\gamma_{j}|C_{k}^{\prime}||C_{j}^{\prime}|}} \left\langle \hat{Y}_{j}^{\prime\,\mathrm{in}\dagger}\hat{Y}_{k}^{\prime\,\mathrm{in}}\right\rangle,\tag{E14}$$

We can expand the expectation value using Eq. (E7) to write everything in terms of the PSDs of the input modes,

$$\left\langle \hat{Y}_{j}^{\prime \, \text{in}\dagger} \hat{Y}_{k}^{\prime \, \text{in}} \right\rangle = \sum_{r,s} w_{jr}^{*} w_{ks} \underbrace{\left\langle \hat{Y}_{r}^{\text{in}\dagger} \hat{Y}_{s}^{\text{in}} \right\rangle}_{\propto \delta_{rs}} \tag{E15}$$

$$=\sum_{r} w_{jr}^* w_{kr} \left\langle \hat{Y}_r^{\mathrm{in}\dagger} \hat{Y}_r^{\mathrm{in}} \right\rangle \tag{E16}$$

$$= w_{j0}^* w_{k0} S_{Y_0^{\text{in}} Y_0^{\text{in}}} + \left(\delta_{jk} - w_{j0}^* w_{k0}\right) S_{\text{vac}},$$
(E17)

where $S_{\text{vac}} = 1/2$ and we have used the unitary relation $\sum_{r=0}^{M-1} w_{jr}^* w_{kr} = \delta_{jk}$ (together with the assumption that the idle input modes all consist of vacuum fluctuations) to move from the second equality to the third equality. The first term in the last equality corresponds to the squeezed-vacuum on the primary (the r = 0) mode, and the second term is a troublesome term which encodes the vacuum fluctuations from the other M - 1 idling input modes. Defining the complex numbers,

$$\Delta_{jk} \equiv e^{i(\varphi_k - \varphi_j)/2} \sqrt{\hbar^2 m_k m_j \Omega_k \Omega_j} \left(\delta_{jk} - w_{j0}^* w_{k0} \right) W_{0j}^* W_{0k},$$
(E18)

and using $S_{\text{vac}} = 1/2$, we can write the force PSD due to shot noise generally as,

$$S_{\text{noise}}^{\text{SN}} \equiv \left| \left(\sum_{k=0}^{M-1} \frac{\mathrm{e}^{\mathrm{i}\varphi_k/2}}{2\chi_k} \sqrt{\frac{\hbar m_k \Omega_k}{\kappa_k |C'_k|}} W_{0k} w_{k0} \right) \right|^2 S_{Y_0^{\mathrm{in}} Y_0^{\mathrm{in}}} + \left(\sum_{j,k} \frac{\Delta_{jk}}{8\chi_k \chi_j^*} \sqrt{\frac{1}{\kappa_k \kappa_j |C'_k| |C'_j|}} \right). \quad (E19)$$

The fluctuations in the amplitude quadrature leads to back-action noise (BA), which is determined by,

$$\left\langle \hat{F}_{j}^{\dagger} \hat{F}_{k} \right\rangle_{\mathrm{BA}} \equiv \\ 8\mathrm{e}^{\mathrm{i}(\varphi_{k}-\varphi_{j})} \sqrt{\hbar^{2} m_{k} m_{j} \Omega_{k} \Omega_{j} \gamma_{k} \gamma_{j} |C_{k}'| |C_{j}'|} \left\langle \hat{X}_{j}'^{\mathrm{in}\dagger} \hat{X}_{k}'^{\mathrm{in}} \right\rangle,$$
(E20)

with

$$\left\langle \hat{X}_{j}^{\prime\,\text{in}\dagger}\hat{X}_{k}^{\prime\,\text{in}}\right\rangle = w_{j0}^{*}w_{k0}S_{X_{0}^{\text{in}}X_{0}^{\text{in}}} + \left(\delta_{jk} - w_{j0}^{*}w_{k0}\right)S_{\text{vac}},\tag{E21}$$

which follows from similar analyses that led to Eq. (E17). We can then write the force noise due to back-action generally as,

$$S_{\text{noise}}^{\text{BA}} \equiv \left\| \left(\sum_{k=0}^{M-1} 2 \mathrm{e}^{\mathrm{i}\varphi_k/2} \sqrt{2\hbar m_k \Omega_k \gamma_k |C'_k|} W_{0k} w_{k0} \right) \right\|^2 S_{X_0^{\text{in}} X_0^{\text{in}}} + \left(\sum_{j,k} 4\Delta_{jk} \sqrt{\gamma_k \gamma_j |C'_k|} |C'_j| \right). \quad (E22)$$

One can likewise find an explicit expression for the quadrature correlation terms by using $\left\langle \hat{Y}_{k}^{\prime \, in\dagger} \hat{X}_{j}^{\prime \, in} \right\rangle = w_{k0}^{*} w_{j0} \left\langle \hat{Y}_{0}^{in\dagger} \hat{X}_{0}^{in} \right\rangle$. From which we can derive the contribution from quadrature correlations to the force PSD,

$$S_{\text{noise}}^{\text{corr}} \equiv 2 \operatorname{Re} \left[\left(\sum_{k=0}^{M-1} \frac{\mathrm{e}^{-\mathrm{i}\varphi_k/2}}{\chi_k^*} \sqrt{\frac{\hbar m_k \Omega_k}{\gamma_k |C_k'|}} W_{0k}^* w_{k0}^* \right) \right. \\ \left. \times \left(\sum_{j=0}^{M-1} \mathrm{e}^{\mathrm{i}\varphi_j/2} \sqrt{\hbar m_j \Omega_j \gamma_j |C_k'|} W_{0j} w_{j0} \right) \tilde{S}_{\hat{X}_0^{\text{in}} \hat{Y}_0^{\text{in}}} \right],$$

$$(E23)$$

where $\hat{S}_{\hat{X}_{\alpha}^{\text{in}}\hat{Y}_{\alpha}^{\text{in}}}$ is defined in Eq. (8) of the main text.

Finally, combining Eqs. (E13), (E19), (E22), and (E23), we derive a general expression of the force noise for the array,

$$\bar{S}_{F_{\text{noise}}}^{(M)} = \left\| \left(\sum_{k=0}^{M-1} \frac{\mathrm{e}^{\mathrm{i}\varphi_{k}/2}}{2\chi_{k}} \sqrt{\frac{\hbar m_{k}\Omega_{k}}{2\gamma_{k}|C_{k}'|}} W_{0k}w_{k0} \right) \right\|^{2} \bar{S}_{Y_{0}^{\text{in}}} + \left\| \left(\sum_{k=0}^{M-1} 2\mathrm{e}^{\mathrm{i}\varphi_{k}/2} \sqrt{2\hbar m_{k}\Omega_{k}\gamma_{k}|C_{k}'|} W_{0k}w_{k0} \right) \right\|^{2} \bar{S}_{X_{0}^{\text{in}}} + 2\operatorname{Re} \left[\left(\sum_{k=0}^{M-1} \frac{\mathrm{e}^{-\mathrm{i}\varphi_{k}/2}}{\chi_{k}^{*}} \sqrt{\frac{\hbar m_{k}\Omega_{k}}{\gamma_{k}|C_{k}'|}} W_{0k}^{*}w_{k0}^{*} \right) \left(\sum_{j=0}^{M-1} \mathrm{e}^{\mathrm{i}\varphi_{j}/2} \sqrt{\hbar m_{j}\Omega_{j}\gamma_{j}|C_{k}'|} W_{0j}w_{j0} \right) \tilde{S}_{\hat{X}_{0}^{\text{in}}\hat{Y}_{0}^{\text{in}}} \right] + \sum_{k=0}^{M-1} |W_{0k}|^{2} \left(4m_{k}\gamma_{k}K_{B}T_{k} \right) + \bar{S}_{\mathrm{res}}^{(M-1)}, \quad (E24)$$

where

$$\bar{S}_{\text{res}}^{(M-1)} = \sum_{j,k} \Delta_{jk} \left(\frac{1}{8\chi_k \chi_j^*} \sqrt{\frac{1}{\kappa_k \kappa_j |C_k'| |C_j'|}} + 4\sqrt{\gamma_k \gamma_j |C_k'| |C_j'|} \right) \\ = \sum_{k=0}^{M-1} |W_{0k}|^2 \left(\frac{\hbar m_k \Omega_k}{16\gamma_k |C_k'| |\chi_k|^2} + 4\hbar m_k \Omega_k \gamma_k |C_k'| \right) - \left| \left(\sum_{k=0}^{M-1} \frac{\mathrm{e}^{\mathrm{i}\varphi_k/2}}{4\chi_k} \sqrt{\frac{\hbar m_k \Omega_k}{\gamma_k |C_k'|}} W_{0k} w_{k0} \right) \right|^2 \\ - \left| \left(\sum_{k=0}^{M-1} 2\mathrm{e}^{\mathrm{i}\varphi_k/2} \sqrt{\hbar m_k \Omega_k \gamma_k |C_k'|} W_{0k} w_{k0} \right) \right|^2. \quad (E25)$$

In Eq. (E24), the first three terms are due to the shot noise, back-action noise, and quadrature correlations, respectively, of the non-trivial input mode, \hat{a}_0^{in} ; these generalize the single-sensor, quadrature noises of Eq. (7) to an array of sensors. The fourth term is the weighted average of the independent mechanical fluctuations of the various sensors. The final term [written out explicitly in Eq. (E25)] contains the residual vacuum fluctuations from the M - 1 idling input modes. These residual vacuum fluctuations do not contribute much to the noise, so long as the resonance frequencies of the mechanical systems are nearly identical, $\Omega_k \approx \Omega \forall k$; see below for more discussion on this.

From Eq. (E24), we can assess the performance of the array for any input radiation to the \hat{a}_0^{in} mode. For instance, the SQL for the array is readily obtained by setting the input to vacuum noise. Doing so, it is straightforward to show that the SQL noise for the array is a weighted average (with respect to the distribution, $|W_{0k}|^2$) of the SQL noises for the individual sensors. Similarly, when squeezing is present in the input field, an expression for the noise in terms of the squeezing parameters can be found, from which we can, e.g., derive a formula for the squeezing angle that approximately cancels anti-squeezing noise; see the next section for details on squeezing. Finally, for a set of identical sensors, it is easy to show that Eq. (E24) reduces to the single-sensor noise of Eq. (7) in the main text (i.e., the residual vacuum fluctuations vanish identically), upon taking $w_{k0} = W_{0k}^*$ and $|W_{0k}| = 1/\sqrt{M}$.

a. Residual vacuum fluctuations We make a few comments about power distribution and the residual vacuum fluctuations from idling input modes of the array [i.e., $\bar{S}_{res}^{(M-1)}$ of Eq. (E25)]. We can choose the distribution weights, $\{w_{k0}\}$, to minimize the residual vacuum fluctuations, but in general, we can not eradicate the vacuum noise entirely. The reason being that, in Eq. (E25), there is a phase difference between the third term and the fourth term—specifically due to the phase of the complex mechanical susceptibility, $\arg(\chi_k)$. This generally restricts us from eliminating the residual shot noise and back-action noise simultaneously [first and second terms in Eq. (E25), respectively, however we can get rid of one or the other by choosing w_{k0} appropriately, ultimately leaving some small amount of residual noise which depends on the phases $\arg(\chi_k)$. On the one hand, this is not much of an impediment near resonance, since $\arg \chi_k(\Omega) = \pi/2 \,\forall k \text{ nor is it an issue far off resonance,}$ where the susceptibility is approximately real. Thus, in these regimes, the residual vacuum noises can be approximately canceled. This, of course, assumes that the resonance frequencies of the mechanical systems are almost identical, $\Omega_k \approx \Omega \,\forall \, k$.

4. Squeezing the array

Assuming that the primary input mode to the array is squeezed (see Fig. 1), such that Eqs. (D18)-(D19) are satisfied, the force noise of Eq. (E24) can be written in terms of the squeezing strength, r, and squeezing angle, θ ,

$$\bar{S}_{F_{\text{noise}}}^{\text{sqz},(M)} = \frac{1}{2} \left| \left(\sum_{k=0}^{M-1} \frac{\mathrm{e}^{\mathrm{i}\varphi_{k}/2}}{2\chi_{k}} \sqrt{\frac{\hbar m_{k}\Omega_{k}}{2\gamma_{k}|C_{k}'|}} W_{0k}w_{k0} \right) \cos\theta - \left(\sum_{k=0}^{M-1} 2\mathrm{e}^{\mathrm{i}\varphi_{k}/2} \sqrt{2\hbar m_{k}\Omega_{k}} \gamma_{k}|C_{k}'|} W_{0k}w_{k0} \right) \sin\theta \right|^{2} \mathrm{e}^{-2r} + \frac{1}{2} \left| \left(\sum_{k=0}^{M-1} \frac{\mathrm{e}^{\mathrm{i}\varphi_{k}/2}}{2\chi_{k}} \sqrt{\frac{\hbar m_{k}\Omega_{k}}{2\gamma_{k}|C_{k}'|}} W_{0k}w_{k0} \right) \sin\theta + \left(\sum_{k=0}^{M-1} 2\mathrm{e}^{\mathrm{i}\varphi_{k}/2} \sqrt{2\hbar m_{k}\Omega_{k}} \gamma_{k}|C_{k}'|} W_{0k}w_{k0} \right) \cos\theta \right|^{2} \mathrm{e}^{2r} + \sum_{k=0}^{M-1} |W_{0k}|^{2} \left(4\hbar m_{k}\gamma_{k}K_{B}T_{k}\right) + \bar{S}_{\mathrm{res}}^{(M-1)}, \quad (E26)$$

where a specific form of the residual vacuum noises, $\bar{S}_{res}^{(M-1)}$, is in Eq. (E25). We can choose the squeezing angle, θ , to approximately cancel the anti-squeezed noise (the term proportional to e^{2r}),

$$\tan \theta_{\omega}^{\star} = -\frac{\left|\sum_{k=0}^{M-1} 8\mathrm{e}^{\mathrm{i}\varphi_{k}/2} \sqrt{\hbar m_{k}\Omega_{k}\gamma_{k}|C_{k}'|}W_{0k}w_{k0}\right|}{\left|\sum_{k=0}^{M-1} \mathrm{e}^{\mathrm{i}\varphi_{k}/2} \frac{\Omega_{k}^{2}-\omega^{2}}{\Omega_{k}} \sqrt{\frac{\hbar m_{k}\Omega_{k}}{\gamma_{k}|C_{k}'|}}W_{0k}w_{k0}\right|},\tag{E27}$$

where we have made explicit that this optimal squeezing angle is frequency dependent. For identical sensors (or a single sensor), this expression reduces to that found in standard texts; see, e.g., Section 5.4.2 in Ref. [39].

5. Classical limits and SQL

If we assume initial vacuum fluctuations in all input quadratures, then the total noise at the output is simply an average of each individual sensor noise, weighted by the distribution $|W_{0k}|^2$. Concretely,

$$\bar{S}_{F_{\text{noise}}}^{\text{cl}}(\omega) = \sum_{k=0}^{M-1} |W_{0k}|^2 \left(4\hbar m_k \gamma_k K_B T_k\right) + \sum_{k=0}^{M-1} |W_{0k}|^2 \left(\frac{\hbar m_k \Omega_k}{16\gamma_k |C'_k| |\chi_k|^2} + 4\hbar m_k \Omega_k \gamma_k |C'_k|\right)$$

(E28)

where "cl" stands for classical. The SQL for the array is then the average SQL of the sensors. This is found by setting $|C'_k| = 1/8\gamma_k |\chi_k|$ in the above and summing the shot noise and back-action, resulting in (ignoring the mechanical fluctuations momentarily),

$$\bar{S}_{F_{\text{noise}}}^{\text{SQL}}(\omega) \equiv \sum_{k=0}^{M-1} |W_{0k}|^2 \left(\frac{\hbar m_k \Omega_k}{|\chi_k|}\right).$$
(E29)

The total noise is then the noise at the SQL plus the thermal fluctuations of the mechanics.

We note that operating at the each sensor independently at the SQL but allowing for joint post-processing is a distributed classical sensing (DCS) scheme which allows for better scaling with the size of the array than independent sensors. Without squeezing/entanglement, the peak performance of a purely classical setup is set by such a CL-DCS scheme.

6. DQS vs. DCS

Here, we briefly elaborate on the differences between distributed quantum sensing (DQS) and distributed classical sensing (DCS) schemes; see Fig. 7. In the DQS scheme (Fig. 7, left), an input entangled state of Mmodes is prepared by splitting a squeezed vacuum state of N_s photons between the modes. The entangled radiation is then distributed to an array of M mechanical oscillators, after which joint post-processing on the signals occurs. In the DCS scheme with squeezed light (Fig. 7, right), M squeezed vacuum—each with N_s number of photons—are generated and independently distributed to M mechanical oscillators, after which joint post-processing on the signals occurs. These two schemes are equivalent in terms of their performance (quantified via, e.g., the SNR or the integrated sensitivity), however the former DQS scheme demands only N_s/M photons per sensor, while the latter DCS scheme demands N_s squeezed photons per sensor. The essential difference between these two setups is that the former utilizes CVMP entanglement to correlate the shot-noise and radiation pressure fluctuations across the sensor array to alleviate the total noise of the mechanics.



Figure 7. DQS versus DCS. (Left) A distributed quantum sensing (DQS) scheme, where a squeezed vacuum of N_s photons is distributed through a passive linear network—thus generating an entangled state—to an array of M mechanical oscillators. The laser power is ME_0^2 . (Right) A distributed classical sensing (DCS) scheme with squeezing, where M squeezed vacuum—each with N_s squeezed photons—impinge on an array of M mechanical oscillators. The power per laser is E_0^2 (i.e., the total power is ME_0^2). These two setups have equal performance, but the DQS scheme consumes only N_s/M squeezed photons per sensor while the DCS scheme consumes N_s squeezed photons per sensor.